AMENDMENTS TO THE SPECIFICATION

Please amend paragraph [0002] as follows:

This fluid dynamic bearing apparatus is for use in spindle motors for disk apparatuses, polygon scanner motors for laser beam printers (LBP) and in other small motors. Enhanced speed, cost reduction, noise reduction, etc., as well as high rotational accuracy, are required for these motors. One of the components which determine these required performances is a bearing which supports spindles of said the motors. In recent years, the use of a fluid dynamic bearing having excellent characteristics for the above required performance has been considered or actually used. For example, in a spindle motor of a disk drive unit such as HDD, a fluid dynamic bearing apparatus which uses a fluid dynamic bearing for each of a radial bearing portion which supports a shaft member in the radial direction and a thrust bearing portion which supports the shaft member in the thrust direction is used. In this fluid dynamic bearing apparatus, hydrodynamic grooves are provided as a hydrodynamic pressure producing means on the inner circumferential surface of a bearing sleeve which forms the radial bearing portion or on the outer circumferential surface of the shaft member. The hydrodynamic grooves are also provided on both end faces of a flange portion of the shaft member which forms the thrust bearing portion or on the faces opposing these (end faces of the bearing sleeve, end faces of a thrust plate, etc.) (for example, refer to Japanese Unexamined Patent Publication No. 2003-239951).

Please replace the heading "BRIEF SUMMARY OF THE INVENTION" with --SUMMARY OF THE INVENTION-- in line 6 on page 3 of the specification.

Please amend paragraph [0008] as follows:

To solve said achieve this object, a fluid dynamic bearing apparatus according to the present invention comprises a hydrodynamic groove region in which a plurality of hydrodynamic grooves are arranged, a smooth surface opposing the hydrodynamic groove region, and a bearing gap which is formed between the hydrodynamic groove region and smooth surface and in which a fluid dynamic pressure is produced by the relative rotation of a fixed side and a rotation side, and the smooth surface is defined by a step so that its length becomes shorter than that of the hydrodynamic groove region. It should be noted that the term "length" used herein means, when

the directions of the normal of the smooth surface and hydrodynamic groove region coincide with the radial direction of the bearing (radial bearing portion), the length of its smooth surface and hydrodynamic groove region in the axial direction, and when the above direction of the normal coincides with the thrust direction of the bearing (thrust bearing portion), the length of its smooth surface and hydrodynamic groove region in the radial direction.

Please cancel the heading "Industrial Applicability" in line 8 on page 21 of the specification.